

modeling techniques are used to develop vehicle dynamics models that are computationally efficient, accurate, and described by physical parameters. Baseline designs are chosen to be consistent with current practice, whereas vehicle targets are defined commensurate with desired advanced performance.

At the vehicle level, responses \mathbf{R}_v must match desired design specifications \mathbf{T}_v . These responses are assumed to be functions of vehicle design variables \mathbf{x}_v and system responses \mathbf{R}_{s_i} , for $i = 1, \dots, n_s$ systems, i.e., $\mathbf{R}_v = \mathbf{r}_v(\mathbf{x}_v, \mathbf{R}_{s_1}, \dots, \mathbf{R}_{s_{n_s}})$. To determine target values for system responses \mathbf{R}_{s_i} and vehicle design variables \mathbf{x}_v , a minimum deviation optimization problem is formulated as follows:

$$\begin{aligned} & \text{Min} \quad \|\mathbf{R}_v - \mathbf{T}_v\| + \varepsilon_v^R \\ & \text{with respect to} \\ & \mathbf{x}_v, \mathbf{R}_{s_1}, \dots, \mathbf{R}_{s_{n_s}}, \varepsilon_v^R \\ & \text{subject to} \\ & \sum_{i=1}^{n_s} \|\mathbf{R}_{s_i} - \mathbf{R}_{s_i}^L\| \leq \varepsilon_v^R \\ & \mathbf{g}_v(\mathbf{R}_v, \mathbf{x}_v) \leq 0 \\ & \mathbf{h}_v(\mathbf{R}_v, \mathbf{x}_v) \leq 0 \end{aligned}$$

where \mathbf{x}_v is the vector of design variables exclusively associated with the vehicle, \mathbf{R}_v is the vector of vehicle responses, \mathbf{R}_{s_i} is the vector of responses for the i -th system making up the vehicle, ε_v^R is a tolerance variable for coordinating system responses, \mathbf{T}_v is the vector of vehicle design targets or specifications, $\mathbf{R}_{s_i}^L$ is the vector of system response values passed up to the vehicle from the i -th system, and \mathbf{g}_v and \mathbf{h}_v are vector functions representing vehicle design constraints.

Once the optimal values of the system level responses \mathbf{R}_{s_i} , $i = 1, \dots, n_s$, are determined by solving the vehicle-level design problem shown above, they are cascaded down to the system level as target values $\mathbf{R}_{s_i}^U$. At the system level, n_s individual minimum deviation optimization problems are formulated to determine the system design variables \mathbf{x}_{s_i} . System responses are assumed to be functions of system design variables alone, i.e., $\mathbf{R}_{s_i} = \mathbf{r}_{s_i}(\mathbf{x}_{s_i})$, given the two-level hierarchy assumption. The minimum deviation optimization problems for the $i = 1, \dots, n_s$ systems are formulated as follows:

$$\begin{aligned} & \text{Min} \quad \|\mathbf{R}_{s_i} - \mathbf{R}_{s_i}^U\| \\ & \text{with respect to } \mathbf{x}_{s_i} \\ & \text{subject to} \\ & \mathbf{g}_{s_i}(\mathbf{R}_{s_i}, \mathbf{x}_{s_i}) \leq 0 \\ & \mathbf{h}_{s_i}(\mathbf{R}_{s_i}, \mathbf{x}_{s_i}) \leq 0 \end{aligned}$$

The analytical target cascading process iterates between the vehicle- and system-level design problems. At each iteration, values of $\mathbf{R}_{s_i}^L$ and $\mathbf{R}_{s_i}^U$ determined at the system and vehicle levels, respectively, are passed up and down to the other level design problem(s).

For target cascading, the models are decomposed into an integrated vehicle model at the top level and four copies of a suspension system model at the low level, see Figure B. The top level vehicle model predicts the vehicle responses \mathbf{R}_v , whereas the suspension model predicts the system responses \mathbf{R}_{s_i} , $i = 1, 2, 3, 4$.

Two sets of targets were used for a new concept truck design study. In the first part of the study, the goal was to achieve a fuel efficiency that would be better by at least 50% than the baseline. Fuel economy was computed at both the Gross Vehicle Weight (GVW, truck weight plus payload) and the Gross Combined Vehicle Weight (GCVW, GVW plus trailer). Performance metrics were to be maintained at the same levels as those of the baseline. In the second part of the study, specific numerical values were not defined as targets; instead, it was attempted to improve all metrics as much as possible.

Attributes of interest for both the series hybrid and the electric drive are compared in Figure C for the first set of targets. Results are normalized such that 1.0 represents the target values. The first three targets are achieved. The last three responses are improved compared to the baseline design but do not achieve the targets. The maximum speed

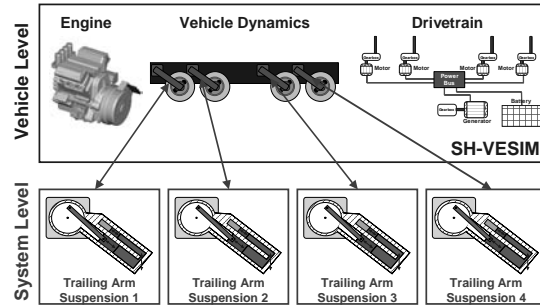


Figure B: Hierarchy of models for target cascading

degradation for the series hybrid configuration comes very close to the target. Note that responses that achieve or exceed their targets become “neutral” to the optimizer, i.e., they do not contribute to the objective

Thus, responses estimated for GVW achieve their targets, while responses estimated for GCVW do not. One can deduce that performance targets at GCVW were too stringent. The same trend is observed for the second set of targets presented in Figure D. Results are normalized with respect to the first set of targets for the sake of comparison. In this case the optimizer tries to improve all responses as much as possible without limits. This leads to a dramatic increase of the ride quality. Hybrid vehicle fuel economy shows more modest, but tangible further improvements over the conventional at both GVW and GCVW. Performance metrics of the hybrid are maintained at about the same level as in the previous case. For the electric drive propulsion system only the ride quality displays significant further improvement compared to the optimization performed for the specific targets, while all other responses remain about the same.

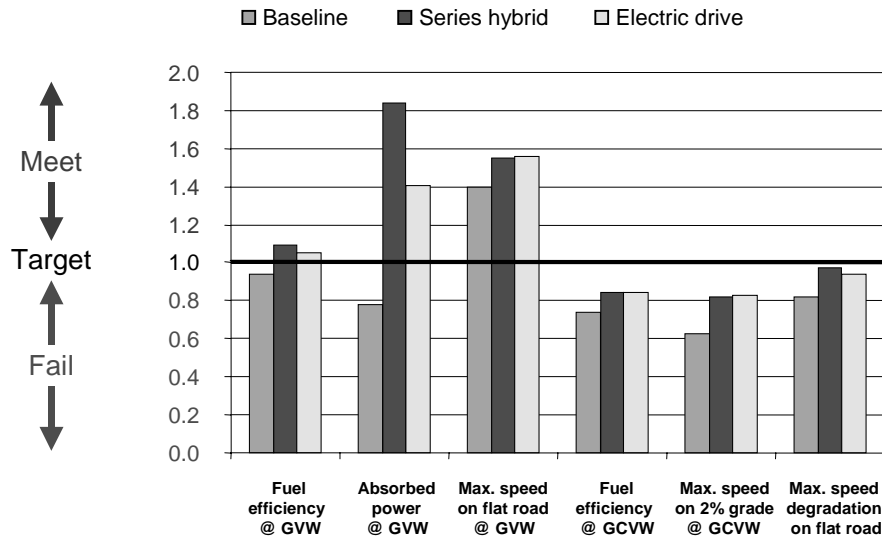


Figure C: Achievement of specific targets

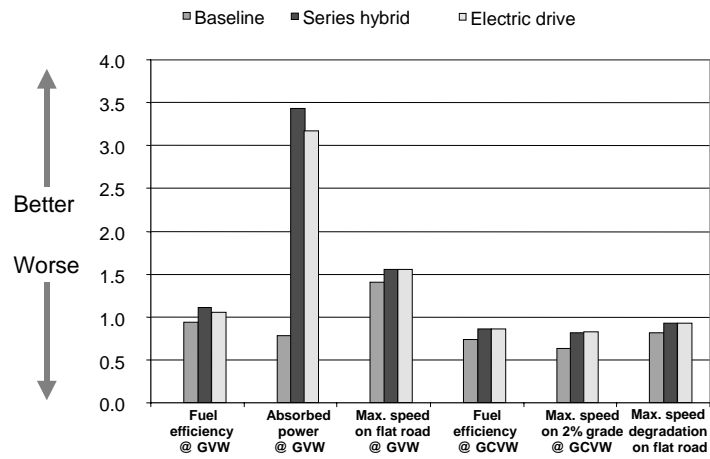


Figure D: Achievement of maximal improvement targets